

* Laws:1 Major diameter:

$$D_{\text{major}} = R_f - R_i \quad [\text{Hand tools \& Abbe vertical}]$$

$$\rightarrow \text{Using B.Gs } (R_i = R_{\text{B.G.1}} + R_{\text{B.G.2}} + R_{\text{zero}} + R_{\text{anvils}})$$

$$D_{\text{major}} = R_2 - R_1 \quad [\text{Microscope \& projector}]$$

2 Minor diameter:

$$R = R_2 - (R_1 - R_{\text{B.G.}})$$

$$\psi = \tan^{-1} \left(\frac{\text{Lead std}}{\pi D_{\text{eff}}} \right) \rightarrow \text{Using B.Gs } (R_1 = R_{\text{B.G.}} + R_{\text{zero}} + R_{\text{anvils}})$$

[Hand tools]

$$D_{\text{minor}} = R \cos \psi$$

$$D_{\text{minor}} = R_2 - R_1 \quad (\text{Consider helix angle}) \quad [\text{Microscope \& projector}]$$

3 Effective diameter:

$$D_e = R_2 - R_1 \quad [\text{Pitch dia. anvils}]$$

$$\rightarrow R_1 = R_{\text{anvils}} + R_{\text{zero}}$$

$$d_{\text{best}} = \frac{P_{\text{std}}}{2} \sec \theta_{\text{std}} \rightarrow \text{larger} \quad [3 \text{ wires \& Abbe vertical}]$$

$$d = \frac{R_2 - R_1}{2} \rightarrow R_{\text{zero}} \quad (\text{using wires only})$$

$$S = R_2' - R_1$$

$$D_e = S - d(1 + \cot \theta_{\text{std}}) + \frac{P_{\text{std}} \cot \theta_{\text{std}}}{2}$$

$$A = \frac{d}{T+d} = \frac{d}{S-d}$$

$$e = \frac{\cos \theta_{\text{std}} \cot \theta_{\text{std}}}{2\pi^2} \times \frac{L^2}{d} \times A^2 (1 + A \sin \theta + A^2 \sin^2 \theta)$$

$$c = 0.001 \frac{F^{2/3}}{D_e^{1/3}}$$

$$D_{e \text{ corrected}} = D_e \cdot e + c$$

— $D_e = R_2 - R_1$ (Consider helix angle) [Microscope & projector]

4 - Included angle measurement:

— Using values of eff. diameter and more dw_2 as well

$$\theta = \sin^{-1} \left[\frac{dw_2 - dw_1}{(y_2 - dw_2) - (y_1 - dw_1)} \right] \quad [\text{Hand tools \& Abbe vertical}]$$

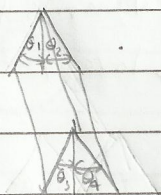
$$\theta_{\text{included}} = 2\theta$$

— Direct measurement [Microscope & projector]

5 - Flank angle measurement:

— Direct measurement and eliminate misalignment error

$$\theta_L = \frac{\theta_1 + \theta_3}{2}, \quad \theta_R = \frac{\theta_2 + \theta_4}{2}$$

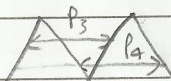
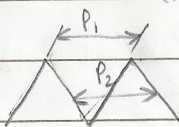


[Microscope & projector]

6 - Pitch measurement:

— $P = R_2 - R_1$ and eliminate misalignment error

$$P = \frac{\sum_{i=1}^4 P_i}{4}$$



7 - Virtual effective diameter:

$$\begin{aligned} \text{V.E.D.} = D_e + \Delta \theta + \Delta P &= s + d(1 + \csc \theta_{\text{std}}) + \frac{P_{\text{std}} \cot \theta_{\text{std}}}{2} \\ &+ 0.0131P(180\rho + 180)^2 + \frac{P}{2}(\cot \phi - \cot \theta) \\ &+ \Delta P_{\text{K\&A}} \cot \theta_{\text{std}} \quad \therefore = 0 \end{aligned}$$

[From std valves]

Problems:*1 Metric thread**

$$D_e = 21.905 \text{ mm}$$

$$P = 3 \text{ mm}$$

$$\Delta P_{\max} = 0.025 \text{ mm}$$

$$\delta\theta_R = 5'$$

$$\delta\theta_L = 6'$$

An external thread of ISO metric form has the following dimensions and errors: Simple effective diameter = 21.905 mm, Pitch = 3.000 mm, Max. Pitch error = 0.025 mm, Right hand flank error = 5 min of an arc and the left hand flank error = 6 min of an arc. Determine the virtual effective diameter of the given thread.

$$\text{Error in flank angle of metric thread} = 0.0131 P_{\text{std}} (\delta\theta_R + \delta\theta_L)$$

$$\Delta\theta = 0.0131 \times 3 (5 + 6) \times \frac{1}{60} = 0.007 \text{ mm}$$

$$\text{Error in pitch} = \Delta P_{\max} \cot \theta_{\text{std}} = 0.025 \cot 30^\circ = 0.043 \text{ mm}$$

$$V.E.D. = D_e + \Delta\theta + \Delta P = 21.905 + 0.007 + 0.043 = 21.955 \text{ mm}$$

2 P = 5 mm, metric thread

$$R_1 = R_{\text{cyl}} - D_{\text{cyl}} = 25.0033 - 15.0030 \\ = 10.0003 \text{ mm}$$

$$R_{\text{avg}} = \frac{\sum_{i=1}^3 R_i}{3} = \frac{13.2007 + 13.2002 + 13.2}{3} \\ = 13.2003 \text{ mm}$$

$$2d = R_{\text{avg}} - R_1 = 13.2003 - 10.0003 \\ = 3.2 \text{ mm}$$

$$Pd = \frac{3.2}{2} = 1.6$$

$$R_{\text{avg}_5} = \frac{64.0272 + 64.027 + 64.0275}{3} = 64.0272 \text{ mm}$$

$$S = R_{\text{avg}_5} - R_1 = 64.0272 - 10.0003 = 54.0269 \text{ mm}$$

$$A = \frac{d}{S - d} = \frac{1.6}{54.0269 - 1.6} = 0.0305$$

$$e = \frac{\cos 30^\circ}{2 \pi^2 \tan 30^\circ} \times \frac{5^2}{1.6} \times (0.0305)^2 \left[1 + 0.0305 \sin 30^\circ + (0.0305 \sin 30^\circ)^2 \right] \\ e = 0.0018 \text{ mm}$$

In measuring the pitch diameter of a 52⁺5 mm metric plug screw gauge the average of the readings taken were:

- Reading over standard cylinder = 25.0033 mm
- Diameter of standard cylinder = 15.0030 mm
- Readings over each of the three wires = 13.2007, 13.2002 and 13.2000 mm
- Readings over thread with the three wires in three different places = 64.0272, 64.0270 and 64.0275 mm.

*Calculate the corrections for rake and compression errors, assuming a measuring force of 2.45 N.

*Calculate the effective diameter, taking into consideration the calculated rake and compression errors.

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$$D_e = S - d(1 + \csc \theta) + \frac{P_{st}}{2} \cot \theta_{st} \\ = 54.0269 - 1.6(1 + \csc 30^\circ) + \frac{5}{2} \cot 30^\circ = 53.5570 \text{ mm}$$

$$C = 0.001 \frac{F_{c,e}^{2/3}}{D_e^{1/3}} = 0.001 \times \frac{(2.45)^{2/3}}{(53.557 + 0.0018)^{1/3}} = 0.00048 \text{ mm}$$

$$D_{corrected} = D_e + e + C = 53.557 - 0.0018 + 0.00048 = 53.556 \text{ mm}$$

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Metric thread

$$P = 1.5 \text{ mm}$$

$$S = 17.019 \text{ mm}$$

$$d = 1.35 \text{ mm}$$

$$\theta_R = 29.5^\circ$$

$$\theta_L = 30.5^\circ$$

$$P = 1.502 \quad 1.496 \quad 1.504 \quad 1.501 \quad 1.5 \quad 1.503 \quad 1.495 \quad 1.502 \text{ mm}$$

$$D_{pitch} = 14.26 \pm 0.035 \text{ mm}$$

Calculate the virtual effective diameter of M16*1.5 screw if the reading over three wires placed over the thread is 17.019 mm. The wires having 1.35 mm diameter, the flank angles of the thread were 29.5° and 30.5° , and the pitch of the thread when checked along the teeth were:

1.502	1.496	1.504	1.501	1.500	1.503	1.495	1.502	mm
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If this screw is to fit in a nut having 14.260 ± 0.035 mm pitch diameter, check whether the screw will fit fully, partially or will not fit at all when engaged with the nut.

$$P_R = P_L = 2.1 - 17.019 = 1.081 \text{ mm}$$

$$S\theta_R = \theta_R - \theta_{std} = 29.5^\circ - 30^\circ = -0.5^\circ$$

$$N\theta_L = \theta_L - \theta_{std} = 30.5^\circ - 30^\circ = 0.5^\circ$$

$$\Delta\theta = 0.0131 P_{st} (1.8 \theta_R + 1.8 \theta_L) = 0.0131 \times 1.5 (0.5 + 0.5) = 0.020 \text{ mm}$$

$$\Delta P_i = P_i - P_{std}$$

$$\Delta P = +0.002 \quad -0.004 \quad +0.004 \quad +0.001 \quad 0 \quad -0.003 \quad -0.005 \quad +0.002 \text{ mm}$$

$$\Delta P_{max} = 0.004 - (-0.005) = 0.009 \text{ mm}$$

$$\Delta P = \Delta P_{max} \cot \theta_{std} = \frac{0.009}{\tan 30^\circ} = 0.016 \text{ mm}$$

$$V.E.D. = D_e + \Delta\theta + \Delta P = S - d(1 + \csc \theta) + \frac{P_{st}}{2} \cot \theta_{std} + \Delta\theta + \Delta P \\ = 17.019 - 1.35(1 + \csc 30^\circ) + \frac{1.5}{2} \cot 30^\circ + 0.02 + 0.016$$

$$V.E.D. = 14.304 \text{ mm}$$

$$V.E.D_{min} = 14.288$$

$$V.E.D_{max} = 14.295 \text{ mm}$$

V.E.D. $>$ V.E.D._{max} \Rightarrow Nut fits partially with 2nd thread

K&A
ANDLOSVA

$$14.304 - 0.016 = 14.288 (1)$$

$$14.288 + 0.002 = 14.29 (1) \checkmark$$

$$14.29 + 0.004 = 14.294 (3) \checkmark$$

$$14.294 + 0.004 = 14.298 (3) \times$$

4-

Metric thread

$$P_{std} = 2 \text{ mm}, D_e = D_0 - 0.966 \times 2P = 12.536 \text{ mm}$$

$$d = 1.35 \text{ mm}$$

$$S.V. = 0.01 \text{ mm}$$

$$R_{range} = 25 \text{ mm}$$

$$R_{zero} = 0.05 \text{ mm}$$

$$All. D_e = 0.00 \pm 0.21 \text{ mm}$$

Three wire set of nominal size 1.35 mm was used to inspect the pitch diameter of M16*2 screw plug. The used external micrometer has a scale value of 0.01 mm, measuring range 25mm, and zero reading of 0.05mm. Knowing that the maximum and minimum permissible allowance of the pitch diameter is +0.21 mm and -0.00 mm, determine the extreme readings of the micrometer such that the readings correspond to the accepted screw lies-in-between.

$$R = S + \text{zero error} + \text{allowance}$$

$$= D_e + d(1 + \cot \theta_{std}) - \frac{P}{2} \cot \theta_{std} + R_{zero} + all.$$

$$= 12.536 + 1.35 \left(1 + \frac{1}{\sin 30^\circ}\right) - \frac{2}{2 \tan 30^\circ} + 0.05 + 0.21$$

$$= 14.934 \text{ mm}$$

$$R_{min} = 14.934 \text{ mm}$$

$$R_{max} = 15.114 \text{ mm}$$

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On a T.M.M. the pitch of a screw was measured and the readings (mm) were as follows:

1 st Side	5.02	7.04	9.05	11.03	13.00	14.98	17.02	19.04	21.00	23.03
2 nd Side	6.01	8.03	10.05	12.05	14.02	16.05	18.03	20.05	22.07	24.07

$$P_{std} = \frac{H}{\text{no. of readings}} = \frac{10}{10} = 1 \text{ mm}$$

If the screw is a perfect screw in other parameters and it'll fit a nut (perfect nut) of 10 mm height. Will this screw engages fully with the nut or partially, and if partially how many teeth will be engaged? Given that the permissible allowance of the pitch diameter is $\pm 0.06 \text{ mm}$.

$$P = R_2 - R_1$$

$$P(\text{mm}) \quad 0.99 \quad 0.99 \quad 1 \quad 1.02 \quad 1.02 \quad 1.07 \quad 1.01 \quad 0.99 \quad 1.07 \quad 1.04$$

$$P_{av} = \frac{\sum P_i}{10} = 1.02 \text{ mm}$$

$$\Delta P_i = P_i - P_{std} = P_i - 1$$

$$\Delta P(\text{mm}) \quad -0.01 \quad -0.01 \quad 0 \quad +0.02 \quad +0.02 \quad +0.07 \quad +0.01 \quad -0.01 \quad +0.07 \quad +0.04$$

$$\Delta P_{max} = +0.07 - (-0.01) = 0.08 \text{ mm}$$

$$\Delta P = \Delta P_{max} \cot \theta_{std} = 0.08 \cot 30^\circ = 0.139 \text{ mm (Metric thread)}$$

$$\Delta P > \text{Allowance} \Rightarrow \text{Nut will fit partially}$$

$$\text{Tooth no.} \quad 1 \quad 2 \quad 3 \quad 4 \quad 5 \quad 6$$

$$\Delta P_{accumulated}(\text{mm}) \quad 0.01 \quad 0.02 \quad 0.03 \quad 0.04 \quad 0.06 \quad 0.09$$

5 tooth only fit from nut